

# Vibration Analyst Category II Equations

## FORCES

### Mass Unbalance

$$F = Me \left( \frac{2\pi N}{60} \right)^2$$

$$M = W/g$$

W = weight of rotor or balance weight, lb

e = rotor eccentricity or radius of balance weight, in

g = gravitational constant, 386.1 in/s<sup>2</sup>

N = RPM

### Spring Force

$$F = Kx$$

K = stiffness of spring, lb/in

x = relative deflection, in

### Damping Force

$$F = C \dot{x}$$

C = damping constant, lb-s/in

$\dot{x}$  = relative velocity

### Inertia Force

$$F = M \ddot{x}$$

M = mass, lb-s<sup>2</sup>/in

$\ddot{x}$  = acceleration, in/s<sup>2</sup>

## MOTIONS

### Velocity (in/s)

$$V = D(2\pi f)$$

D = peak displacement, in

f = frequency, cycles/s (CPS)

$\pi = 3.14$

*Copyright © 2011 by the Vibration Institute. All rights reserved. No part of this document may be reproduced without express written permission of the Vibration Institute.*

## Acceleration

$$A = V(2\pi f)$$

A = acceleration, in/s<sup>2</sup>  
1 g = 386.1 in/s<sup>2</sup>

## FREQUENCIES

### Bearing Frequencies

$$FTF = \left(\frac{\Omega}{2}\right) \left[1 - \left(\frac{B}{P}\right) \cos CA\right]$$

$$BPFI = \left(\frac{N}{2}\right) \Omega \left[1 + \left(\frac{B}{P}\right) \cos CA\right]$$

$$BPFO = \frac{N}{2} \Omega \left[1 - \left(\frac{B}{P}\right) \cos CA\right]$$

$$BSF = \left(\frac{P}{2B}\right) \Omega \left[1 - \left(\frac{B}{P}\right)^2 \cos^2 CA\right]$$

FTF = fundamental train frequency  
BPFI = ball pass frequency, inner race  
BPFO = ball pass frequency, outer race  
BSF = ball spin frequency  
RPM = shaft speed

CA = contact angle  
 $\Omega$  = machine speed  
N = number of rolling elements  
P = pitch diameter, in  
B = ball or roller diameter, in

Bearing defect frequencies are same units as machine speed

### General Guideline Bearing Frequencies

(for use in FMax selection ONLY)

$$BPFO = 0.41 \times \text{RPM} \times N$$

$$BPFI = 0.59 \times \text{RPM} \times N$$

$$FTF = 0.41 \times \text{RPM}$$

$$BSF = 0.22 \times \text{RPM} \times N$$

*Copyright © 2011 by the Vibration Institute. All rights reserved. No part of this document may be reproduced without express written permission of the Vibration Institute.*

### **Natural Frequency**

$$f_n = \frac{1}{2\pi} \sqrt{\frac{k}{m}}$$

k = stiffness, lb/in

m = w/g

w = weight, lb

g = gravitational constant, 386.1 in/s<sup>2</sup>

f<sub>n</sub> = natural frequency of a single-degree-of-freedom system, Hz

### **Roll Frequency**

$$f = \frac{V}{5\pi D}$$

V = web velocity, ft/min

D = roll diameter, in

f = frequency, Hz

## **SIGNAL PROCESSING**

### **Dynamic Range**

$$\text{dB} = 20 \log \frac{V_m}{V_r}$$

$$\frac{V_m}{V_r} = 10^{\frac{\text{dB}}{20}}$$

V<sub>m</sub> = voltage measured

V<sub>r</sub> = voltage reference

dB = decibels

*Copyright © 2011 by the Vibration Institute. All rights reserved. No part of this document may be reproduced without express written permission of the Vibration Institute.*

## **RMS**

peak = 1.414 rms

## **Resolution**

Resolution = (frequency span x window noise factor x 2)/#FFT lines

window noise factor =

1.0 for uniform window

1.5 for Hanning window

3.8 for flat top window

## **Data Acquisition Time (DAT)**

DAT = # FFT lines/frequency span

## **Default Frequency Spans**

Operating Speed	= 10 x RPM
Rolling Element Bearings	= 10 x BPFI
Fluid Film Bearings	= 10 x RPM
Vane/Blade Pass	= 3 x # Vanes/Blades x RPM
Electrical	= 3 x 2X Line Frequency
Gear Mesh	= 3 x Gear Mesh Frequency

*Copyright © 2011 by the Vibration Institute. All rights reserved. No part of this document may be reproduced without express written permission of the Vibration Institute.*